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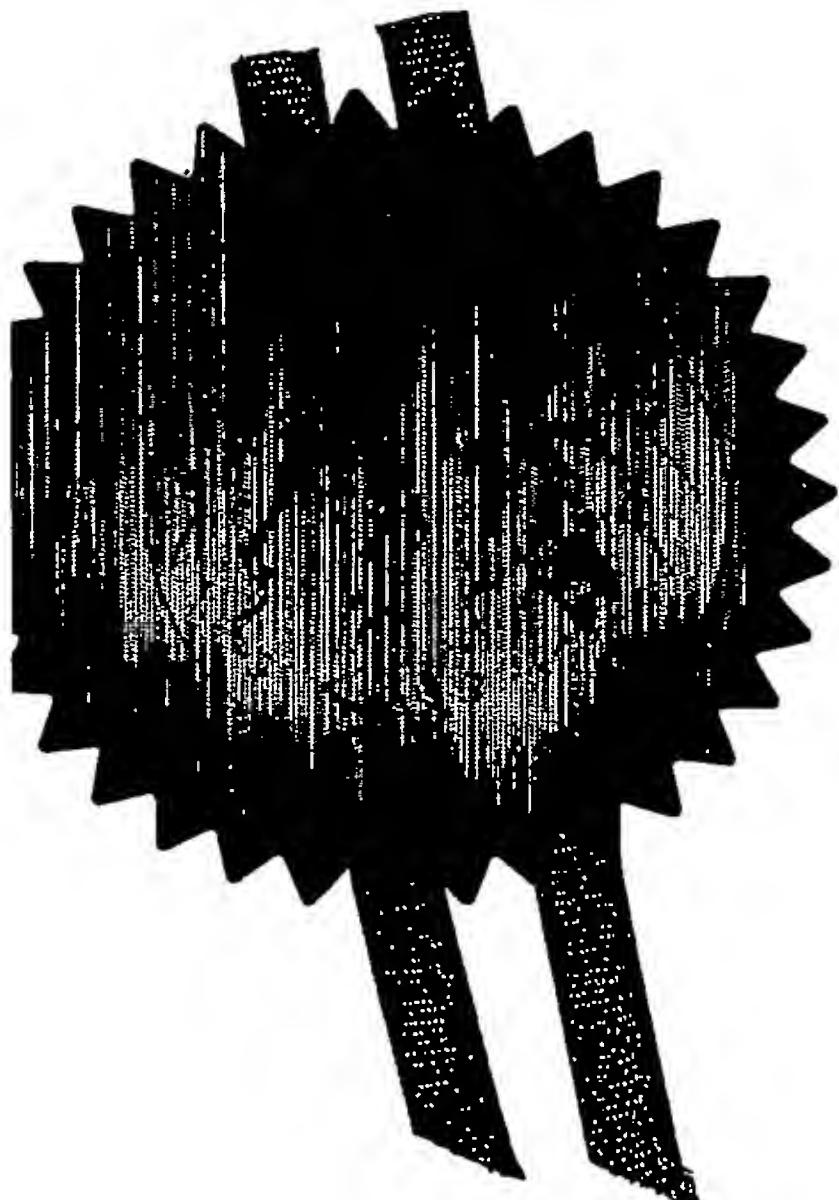
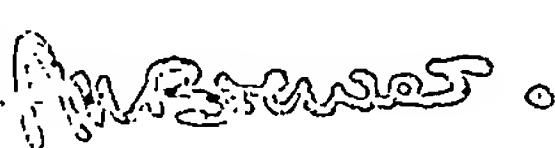
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F072241PGB

0323160.2

F 3 OCT 2003

2. Patent application number

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8726549001

13

If the applicant is a corporate body, give the country/state of its incorporation

Sweden

4. Title of the invention

Air Suspension System

5. Name of your agent (if you have one)

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Description 14

Claim(s) 2

Abstract 1

Drawing(s) 4 *JMK*

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0121 643 5881

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Air Suspension System

The present invention relates to a motor vehicle air suspension system and to an electronically controllable suspension element for use in such system.

In a vehicle air suspension system either rubber air springs or pneumatic dampers, or both, replace a conventional steel spring and damper arrangement. Air suspension systems offer the advantages of improved ride quality and, if electronically controlled, a facility for adjustment of the vehicle ride height (more specifically the distance between the chassis and the axle) under varying load and dynamic conditions. Height control is achieved by variation of the volume of air within the springs/dampers. As the volume increases or decreases, so too does the height of the vehicle body above the ground, assuming no change in payload.

Systems are known that offer 'load levelling', whereby a specific height is maintained as payload varies, or may in addition offer variable ride heights - e.g. increased ground clearance for driving over uneven terrain.

One problem with such air suspension systems is that in order for the ride height to be variable, the air springs need to be softly sprung, having greater freedom of travel (i.e. displacement) than, say, conventional steel springs. This means that the vehicle is susceptible to greater degrees of pitch and roll, especially when braking, accelerating or cornering.

It is known to provide electronic control of air suspension systems. A controller (ECU) can be used to control adjustment of the volume of air in the air springs, or the damping characteristics of the dampers. These systems suffer from the degree of complexity, especially in terms of the number of components and the electrical circuitry and sensors required. It is also necessary to integrate all of the suspension components (air source, valves, air springs and dampers associated with each suspension point - i.e. each wheel or-axle), into a system controlled by the ECU.

It is an aim of the present invention to provide an improved air suspension system which substantially alleviates the aforementioned problems.

According to a first aspect of the present invention there is provided a vehicle air suspension system comprising a plurality of suspension elements, wherein each element includes:

- at least one air suspension unit comprising a fluid damper and an air spring;
- height sensing means for providing a ride height signal; and
- a controller responsive to said ride height signal for controlling a damping coefficient of said fluid damper and a volume of air in said air spring.

In a preferred embodiment, each suspension element is associated with one wheel of the vehicle. For a four-wheel vehicle, each suspension element may be one associated with each of all four wheels of the vehicle or each of the two rear wheels only. Each element may comprise a single air suspension unit.

It is an advantage that each suspension element is a fully integrated stand-alone element having its own controller. Thus, each element may be of an identical construction and therefore may be associated with any one of the four wheels (i.e. fitted at any of the four "corners" of the vehicle). This also reduces the overall system complexity (when compared with centrally controlled air suspension systems), is of convenience for replacement and maintenance and reduces the amount of electrical circuitry required. Manufacturing complexity of the suspension elements is also reduced because the same suspension elements are used for front and rear wheels. Vehicle assembly times are also reduced because the same elements are fitted to all four wheels.

Preferably, the controller is responsive to further signals indicative of prevailing conditions of the vehicle. The further signals may include signals indicative of any or all of: vehicle speed; foot brake position; lateral acceleration; engine (running/not running); gear selector position; pressure of air in the air springs. The controller may also receive input signals from push buttons or switches within the vehicle cabin.

It is an advantage of the system that independent control of the characteristics of each suspension element enables the system to compensate for variations in the driving conditions as well as allowing for variations in ride height.

The controller may include a programmable microcontroller to enable 'tuning' of the unit in accordance with the required height settings and control strategy for the vehicle.

It is a further advantage that the suspension element can be adapted for use on different vehicles by reprogramming of the programmable controller.

According to a second aspect of the present invention there is provided a vehicle air suspension unit comprising:

a fluid damper;
an air spring;
height sensing means for providing a ride height signal; and
a controller responsive to said ride height signal for controlling a damping coefficient of said fluid damper and a volume of air in said air spring.

In a preferred embodiment, the height sensing means comprises a sensor enclosed within the air spring. This protects the sensor by providing an environmental seal around it.

Embodiments of the invention will now be described by way of example with reference to the following drawings, in which:

Figure 1 shows a configuration of an air suspension unit in accordance with one aspect of the invention;

Figure 2 shows an arrangement of air suspension units associated with the wheels of a four-wheeled vehicle, in a suspension system according to another aspect of the invention;

Figure 3 is a block diagram representation of signal inputs and outputs to/from an ECU of the suspension unit of Figure 1;

Figure 4 is a graphical representation of damper settings for the suspension unit of Figure 1;

Figure 5 is an illustration of a vehicle and body movements thereof;

Figure 6 is a diagram showing displacements of a vehicle chassis relative to its axles; and

Figure 7 is a graph showing a displacement response in a "bounce" condition.

Referring to Figure 1, an air suspension unit 10 comprises a fully-integrated assembly for mounting between the underside of a vehicle chassis and a wheel axle. The unit 10 has a fluid damper 12 extending through a rubber pneumatic air spring 14 that envelops at least a portion of the damper 12.

The fluid damper 12 is of a known telescopic tubular variety having a lower tubular section 16 extending downwardly and coaxially from an upper tubular section 18. The damper 12 has a lower mounting 20 at a lower end of the first tubular section 16, while the second tubular section 18 is fixedly mounted to a top housing 22. A damping fluid is trapped between the tubular sections. Under axial movement between the tubular sections 16, 18, so as to extend or compress the damper, one section slides telescopically within the other and causes the fluid to be forced through a restricted flow path (not shown) within the damper 12. The restricted flow path has a variable restriction, to allow the damping coefficient to be adjusted. An electrically actuated mechanism is provided to effect this adjustment.

The air spring 14 is fixedly mounted at a top end to an underside of a plate 24 forming part of the housing 22, and at a lower end to a mounting 26. The mounting 26 is rigidly attached to, and forms an airtight seal around the first tubular section 16 of the damper 12. The air spring 14, together with the mounting 26 and the plate 24 define an

annular cavity 30 around the damper 12, to which compressed air is supplied by way of an electrically actuated valve 32 in the top housing 22. Operation of the valve 32 controls the supply of air to and exhaustion of air out of the air spring 14. When compressed air is supplied to the air spring 14, this causes the air spring 14 to be inflated, and the damper 12 to extend. Similarly the damper 12 will compress when the air spring is deflated. However, this does not affect the damper characteristics, which depend only on the internal orifice size and oil flow path.

A height sensor 33 is provided for sensing and signalling of a ride height. This provides a signal indicative of displacement between the axle and the chassis. The height sensor 33 is mounted inside the air spring 14 and senses displacement between the top of the mounting 26 (or a point on the first tubular portion 16 of the damper 12), and the underside of the plate 24 (or a point on the second tubular portion 18 of the damper 12). The height sensor 33 may be of any type suitable for providing a signal indicative of displacement. One example is a linear variable differential transducer (LVDT), which produces a voltage output indicative of displacement of a ferrous core member relative to an induction coil. The height sensor 33 shown in Figure 1 is enclosed inside the air spring 14. This provides an air-tight seal around the sensor and provides environmental protection (e.g. from contamination or corrosion).

An electronic control unit (ECU) 36 is also housed within the top housing 22. Various signal inputs, including the ride height signal are provided to the ECU 36, which in response sends appropriate output signals to control the air suspension unit 10. The ECU 36 controls the electrical actuation of the 'damping coefficient' adjustment of the damper 12, and the air volume in the air spring 14 by actuation of the valve 32. The ECU may also control operation of an air compressor and reservoir for supply of compressed air to the air spring 14. Electrical power and communications signals are supplied to the ECU 36 by way of a connector 34. The only other connections to the unit 10 are made to an air supply by way of a pneumatic connector 38, and to an electrical power source by way of a power connector 40 to provide power for activating the pneumatic valve 32.

The ECU 36 may include a programmable microcontroller to allow adjustment of the damping coefficient and the control of the valve 32 to be tuned in accordance with the required height settings and control strategy for the vehicle. The tuning allows optimisation of the behaviour of the air suspension unit 10 for a particular vehicle's requirements.

In use, the suspension unit 10 is mounted between an axle and the chassis of the vehicle. The unit 10 may be one associated with each of all four wheels of the vehicle or each of the two rear wheels only. Each damper 12 is controlled independently of the associated air spring 14, and each suspension unit 10 is controlled independently from each other suspension unit on the vehicle. This independent control allows for improved handling and performance, as will be evident from the discussion below.

The ECU 36 receives input signals directly from the height sensor 33 within the same unit 10. The ECU 36 also receives signals from the ECUs of other suspension units on the vehicle, and from various other sensors and control units on the vehicle. Examples of other signals that may be received include the status of: the vehicle speed; the foot brake position (e.g. applied or not applied); lateral acceleration from an accelerometer; the engine (e.g. running or not running); the gear selector position (e.g. in the case of a vehicle with automatic transmission: 'Park', 'Ride', 'Neutral', 'Drive', 'Low'); the pressure of the air within the springs. The ECU may also receive input signals from push buttons or switches within the vehicle cabin, possibly via an interim control unit.

Air is supplied to the air spring 14 from a source of compressed air such as a pump, either directly or (optionally) from an intermediate reservoir. The compressed air source may be integrated within the air suspension system.

It is necessary to remove moisture from the air supplied to the springs to prevent a build up of liquid in the spring, which could otherwise lead to damage to the valves, pipework or springs caused by ice, should the water freeze. A moisture remover may be integrated within the air suspension unit 10.

Figure 2 is a schematic representation of communication paths for a suspension system on a vehicle. The vehicle has four wheels 111- 114 associated with each of which is an independent air suspension unit 101, 102, 103, 104. Signals (e.g. height sensor signals) from each of the air suspension units 101-104 are passed to each other as shown by the broken lines 120. The transmission of these signals occurs by way of a communications databus linking all of the units 101-104. Signals from other sensors on the vehicle are fed to all the air suspension units via the databus as shown by broken line 122.

Figure 3 shows the interaction of input and output signals from the ECU 36 (as shown in Figure 1), for one air suspension unit 101 (as shown in Figure 2). Input signals include: the height sensor signal 152 from the unit's own height sensor; signals received via a databus 150 including signals from the other air suspension units on the vehicle 154 (including the height sensor signals) and other data signals 156 from the vehicle; and data signals 158 from any other sensors fitted to the unit 101 (for example relating to the pressure of air in the system). The output signals include: a damper control signal 160 for control of the damping coefficient of the damper; an air spring control signal 162; and an output data signal 164 to the other air suspension units 102-104 via the databus 150.

The basic requirements for a vehicle suspension system are twofold:

- (i) Maintenance of contact between the vehicle tyres and the ground over which it is being driven; and
- (ii) Comfort of the driver and passengers, through isolation from 'shock' loads and vibration arising from dynamic road inputs.

The suspension arrangement described herein satisfies these requirements through damper control for optimisation of handling and stability characteristics as dynamic conditions change, and air spring control to provide a high level of driver and passenger comfort under all conditions.

Both the dampers and the air springs influence each of the basic requirements. However the control strategy is based on the facts that damper control has significantly

the greater effect on (i), while (ii) relies more on the air springs than the dampers in this arrangement.

Vehicle suspensions that feature air springs are by nature 'softly' sprung and have significantly greater freedom of travel when compared to conventional steel-sprung assemblies. The latter is necessary if variable ride heights are to be offered, but a corollary is increased roll (see below) for a given degree of lateral acceleration. For the system described, this effect is counteracted via damper control.

For damper control, the most important input to each ECU 36 is that from its associated height sensor. Other inputs from the vehicle databus 150 can be used by the ECU 36 for damper control. These include:

Lateral Acceleration: if the vehicle is fitted with a lateral accelerometer that communicates on the databus, the ECUs may read values from this. Lateral acceleration data may be used instead of or in addition to height sensor signals in the control of roll (see below).

Vehicle speed status: informing each ECU 36 whether or not the vehicle is in motion. If so then priority is given to damper control. If not then focus is given to control of the air springs. In addition, if the rate of change of speed is calculated, then an acceleration-induced pitch condition may be predicted and the damper 12 set appropriately to counteract it. Alternatively, in the case of vehicles with automatic transmission, a gear selector position signal may be used to inform each ECU 36 whether or not the vehicle is in motion.

Footbrake status: pitch is induced by braking (see below), and if the ECU 36 is informed that the brakes are being applied, then a pitch condition can be predicted and the damper 12 can be set appropriately to oppose it. In addition, if information regarding the rate of application of the brake is available, then the magnitude of the pitch condition can also be predicted.

Steering wheel angle: roll is induced by steering inputs from the driver to a vehicle travelling above manoeuvring speeds (see below). If the ECU 36 is given steering wheel angle information, then the magnitude and direction of a roll condition can be predicted and the damper 12 can be set appropriately to oppose it.

For each suspension unit independently, the ECU 36 selects the damper setting most appropriate to the prevailing vehicle conditions as shown in Figure 4. There may, for example, be four such settings of varying degree of compliance or firmness that could be described as 'soft', 'normal', 'sports' and 'firm'. Having only a given number of discrete settings available simplifies both the physical configuration of the system and the control strategy, and means that a relatively inexpensive microcontroller can be used.

Each setting is manifest in the form of a specific oil path within the damper 12. Each specific oil path has an associated damping coefficient so that changing the path changes the response characteristics of the damper 12. The means for changing the oil path is controlled by a signal from the ECU 36.

The control strategy should take account of differing requirements between the front and rear of the vehicle. Modern vehicle suspension systems are usually configured such that the spring rates are lower and the damping is softer at the front than at the rear. This ensures that there is a predictable understeer characteristic, and so the vehicle 'feels natural' to the driver and passengers when cornering or negotiating bends. For these reasons, in the arrangement of this embodiment, the suspension control units at the front of the vehicle are given higher status on the data bus hierarchy than those at the rear. In addition, the control strategy will give priority to the front of the vehicle. The system takes action to correct errors sensed on the front of the vehicle before those sensed at the rear. For suspension units on the same axle of the vehicle, priority is given to whichever has the largest 'error' - i.e. requires the greatest degree of adjustment.

Figure 5 shows the ways in which a suspended vehicle body can move relative to the wheels. Pitch is a relative displacement between the front and rear of a vehicle, this results in a rotation about a transverse axis Y passing through an instantaneous centre that is close to the centre of mass. Roll is a relative displacement between one side of a moving vehicle and the other and results in a rotation about a longitudinal axis X.

Each ECU 36 samples the reading from its associated height sensor at small, predefined intervals ($\leq 10\text{ms}$). In addition, it samples the values from the other height sensors at similar intervals via the communications databus 150. By comparison of height sensor readings, it is possible to detect the occurrence of a pitch or roll condition. Furthermore, because sampling intervals are defined, the rate of progress of the condition can be determined.

A pitch condition is detected by comparison of readings between height sensors on the front and rear axles, i.e. front-right versus rear-right, front-left versus rear-left. Pitch can be induced by changes in speed arising from braking or acceleration. It may also be induced by the loading or unloading of a stationary vehicle, but in such cases the suspension control units would 'know' that the vehicle is stationary from a 'vehicle speed zero' or 'gear selector in Park' signal from the databus 150 and would action the air springs as necessary for re-levelling.

A roll condition is detected by comparison of readings between sensors on the same axle, i.e. rear-right versus rear-left, front-right versus front left. This is depicted in Figure 6. Roll is induced when a vehicle negotiates a bend or is subjected to a strong crosswind.

The 'direction' or 'sign' of the pitch or roll condition can also easily be determined and signifies whether the roll is from left to right or right to left. With reference to Figure 6, for example, if roll is always calculated as $h_1 - h_2$ then for the condition shown it would be negative. If h_1 is greater than h_2 then it would be positive.

The roll angle, Θ , can be calculated simply as $\Theta = (h_1 - h_2) / t$ (radians), where t is the centre to centre distance between the wheels (the 'track').

The rate of change of roll angle can also be readily determined, given the sampling intervals of the signals from the height sensors. This is important because it provides an indication of the severity of the condition, and the speed with which the control units must invoke a response from the dampers and with which this response must actually be put into effect.

Given the degree of pitch or roll, the rate of progress and the sign, the ECU 36 is able to determine the most appropriate setting for its associated damper 12. These settings may be stored within a memory in the ECU 36 as a 'look-up table' of discrete values. Once the most appropriate setting is established, the ECU 36 sends a signal to the damper 12 to provide the required damping coefficient by changing the oil path within the damper 12.

In addition to pitch and roll, the system is able to detect and make corrections for linear motion of the sprung mass of the vehicle in the vertical direction (i.e. along a vertical axis Z as shown in Figure 5). This is generally referred-to as 'bounce', and arises from simultaneous displacement of both wheels on a given axle as a result of traversing an obstacle on the road or an undulating road surface.

Bounce is essentially a two-stroke action: bump - upward motion of the tyres relative to the vehicle body, causing compression of the suspension springs and dampers; and rebound - downward motion of the tyres relative to the vehicle body, causing extension of the suspension springs and dampers.

Figure 7 illustrates a vehicle bump-rebound cycle. The time taken for a complete cycle is t_1 and the reciprocal of this value is the bounce frequency. If the system is to set dampers in order to oppose a bounce condition, then action must be taken early in the bump-rebound cycle. If the time taken for the bump stroke to reach its peak displacement (t_2) is known, and the cycle is assumed to follow the sinusoidal

form as shown, then the frequency f of the bump rebound cycle can be calculated from $f = 1/t_1 = 1/(4 \times t_2)$.

In the system described, height readings are sampled at regular time intervals (approx. 10ms) by each ECU. By comparing each reading with the preceding one, the turning point of the bump stroke can be detected and therefore the time t_2 calculated given the number of sampling intervals that have elapsed since the onset of the displacement. The bounce frequency can then very quickly be calculated and dampers can be set accordingly.

Every vehicle has natural frequencies of vibration. Of particular concern in the control strategy are bounce frequencies at or close to any of the natural frequencies, f_n , in the vertical direction because these would induce undesirable resonance conditions. If the f_n values are known, then they can be stored in the memory of each ECU. If a condition is detected with a calculated frequency close to an f_n value, then opposing action can be taken.

The air spring 14 (see Figure 1) is controlled by signals from the ECU 36 to control operation of the electrically actuated valve 32 to provide more air to the spring 14, or to relieve air from the spring 14. Signals from the ECU 36 may be used to control activation of a pump or associated air valves forming part of the compressed air supply to the system.

The volume of air in the springs may be varied to control the ride height in the following situations.

Load Levelling: the ECU activates the pump and valves as necessary to maintain a preset ride height - the 'Design Ride Height' - as payload conditions vary.

Extended Ride Height: in response to the driver pushing a button in the vehicle cabin, the springs are inflated in order to raise the vehicle to a level above the Design Ride Height. This increases the clearance between the underside of the vehicle and the ground beneath it, which is desirable when driving over rough or uneven terrain.

High Speed Lowering: at relatively high vehicle speeds, say above 90kph (56mph), it can be advantageous to deflate the springs until the vehicle is lowered to a predetermined level below Design Ride Height. This correspondingly lowers the roll centre and centre of mass (see Figure 5), and consequently stability is improved.

'Kneel' Height (Stationary vehicle only): in response to the driver pushing a button in the vehicle cabin, the springs are deflated until the vehicle is lowered to a prespecified level below the Design Ride Height. This facilitates loading or unloading, and also boarding and alighting of passengers in the case of large vehicles (e.g. vans, trucks, sport utility vehicles).

The essential demands of control of the dampers and control of the air springs are very different, and there will be no conflict between changes in damper coefficient and spring air volume.

The dampers, and their associated controls, must react very quickly (e.g. in around 15ms or less) to changing conditions. This is not so in the case of the air springs, where relatively slow reaction to control inputs is sufficient.

In the event of a malfunction, the system should 'fail safe'. If control of the dampers and/or the air springs is lost, the suspension should be left in such a condition that the vehicle remains stable. If power is lost, for example, the dampers would be left set at 'firm' and air would be 'locked-in' to the springs. In the event of deflation of one or more of the air springs, the associated damper would be set 'soft' to minimise shock or vibrational inputs into the vehicle chassis.

If the system is fitted to a vehicle having antilock braking (ABS), then it could provide assistance to the ABS control strategy. An air suspension maintains the vehicle height within a given band about a preset ride height, often referred-to as the 'design ride height' or 'trim' height, as payload changes. This operation is known as 'load levelling' and is normally only applied to a static vehicle. Deviations from trim height are detected by the height sensors and signalled to the ECU. Where a static vehicle is

lowered by the addition of payload, whether passengers or freight, then the displacement will be directly related to the magnitude of the payload. A 'look-up table' of change in height displacement against change in payload could be stored by the ECU. Information regarding payload changes can be provided to the ABS, which can then optimise braking strategy accordingly for minimisation of overall stopping distance. For instance, a greater payload ideally requires application of a greater braking force in a shorter time interval.

Claims

1. A vehicle air suspension system comprising a plurality of suspension elements, wherein each element includes:
 - at least one air suspension unit comprising a fluid damper and an air spring;
 - height sensing means for providing a ride height signal; and
 - a controller responsive to said ride height signal for controlling a damping coefficient of said fluid damper and a volume of air in said air spring.
2. A suspension system according to claim 1 wherein each suspension element is associated with one wheel of the vehicle, whereby for a four-wheel vehicle, each suspension element is associated with each of all four wheels of the vehicle, or each of two rear wheels only.
3. A suspension system according to claim 2, wherein each element comprises a single air suspension unit.
4. A suspension system according to any preceding claim, wherein each element is of an identical construction.
5. A suspension system according to any preceding claim, wherein the controller is responsive to further signals indicative of prevailing conditions of the vehicle.
6. A suspension system according to claim 5, wherein the further signals include signals indicative of any or all of: vehicle speed; foot brake position; lateral acceleration; engine (running/not running); gear selector position; and pressure of air in the air springs.
7. A suspension system according to claim 5 or claim 6, wherein the controller also receives input signals from push buttons or switches within the vehicle cabin.
8. A suspension system according to any preceding claim, wherein the controller includes a programmable microcontroller to enable 'tuning' of the unit in accordance with the required height settings and control strategy for the vehicle.

9. A vehicle air suspension unit comprising:
 - a fluid damper;
 - an air spring;
 - height sensing means for providing a ride height signal; and
 - a controller responsive to said ride height signal for controlling a damping coefficient of said fluid damper and a volume of air in said air spring.
10. An air suspension unit according to claim 9, wherein the height sensing means comprises a sensor enclosed within the air spring.

AbstractAir Suspension System

The invention relates to a vehicle air suspension system and an air suspension unit therefore. The air suspension system comprises a plurality of suspension elements, wherein each element includes at least one air suspension unit. The air suspension unit comprises a fluid damper and an air spring; a height sensing means for providing a ride height signal; and a controller responsive to the ride height signal for controlling a damping coefficient of the fluid damper and a volume of air in the air spring.

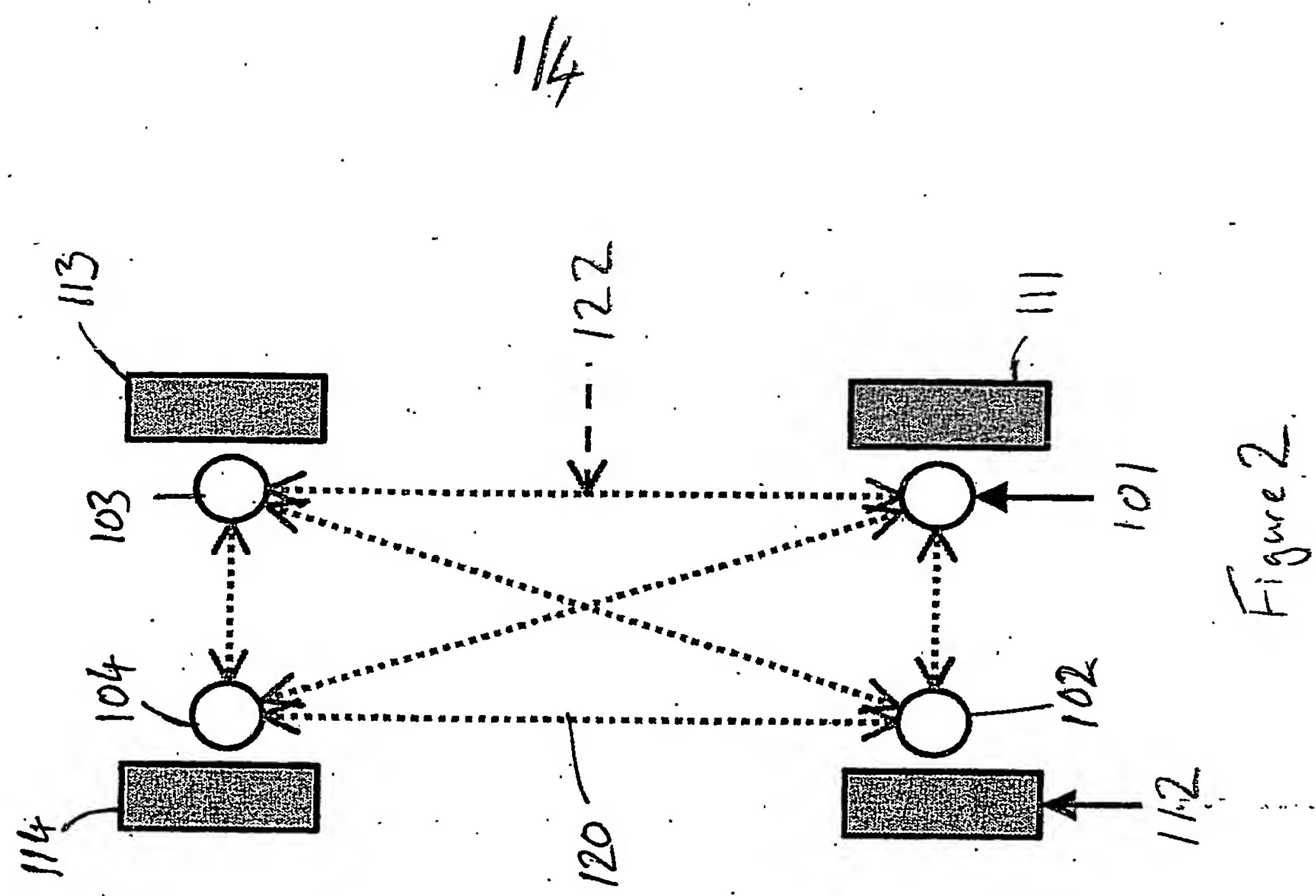


Figure 2

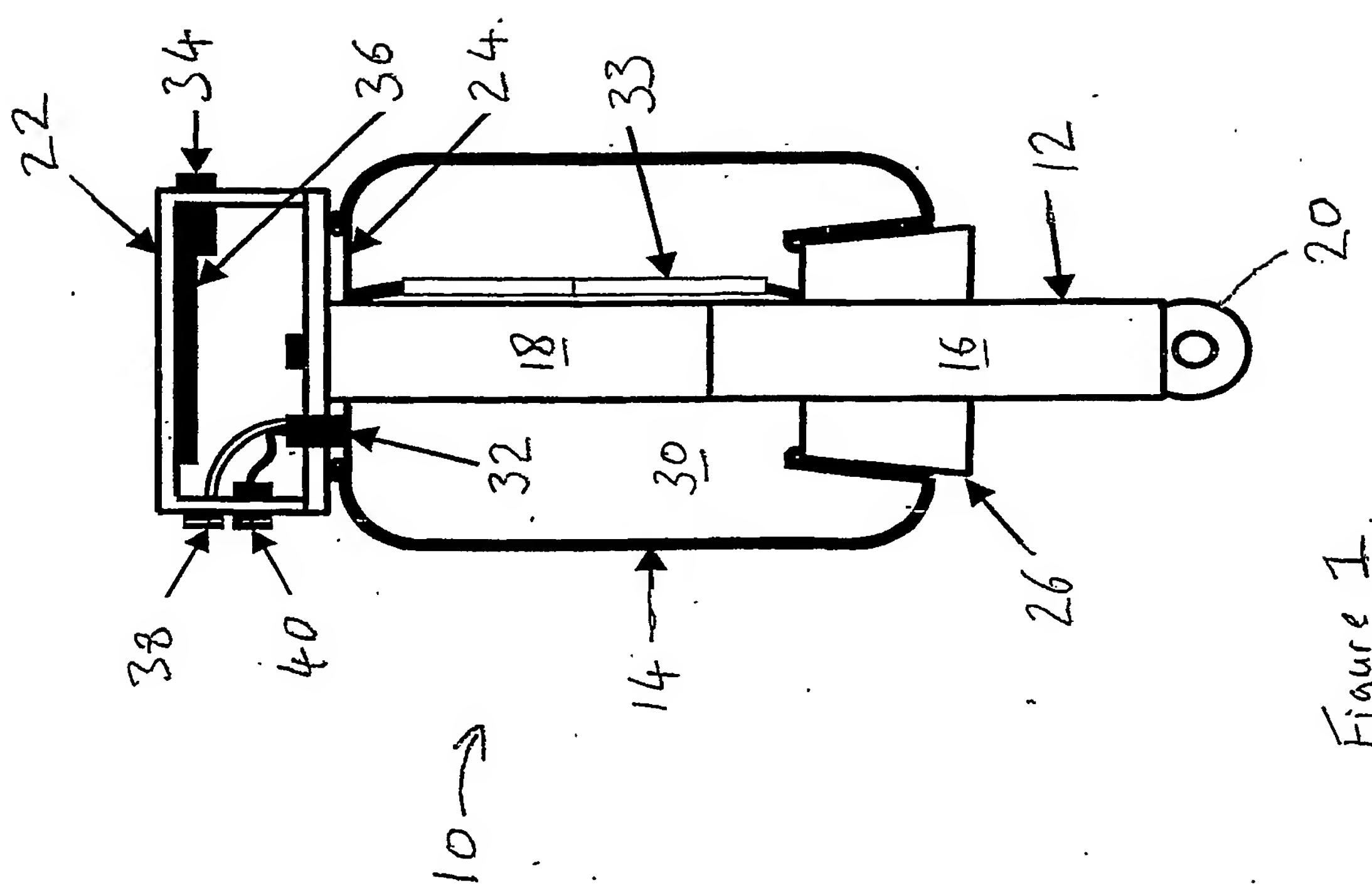


Figure 1

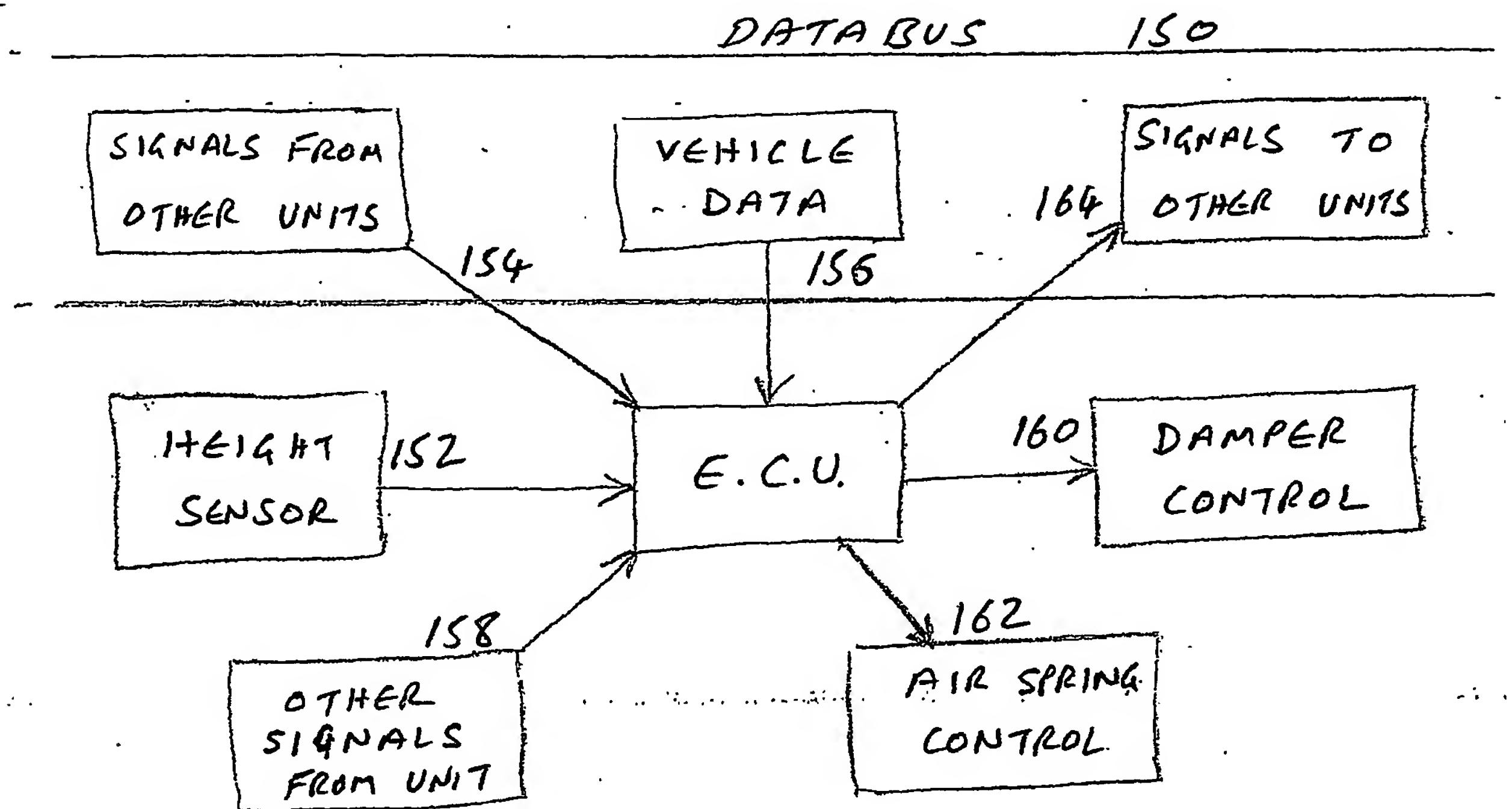


Figure 3

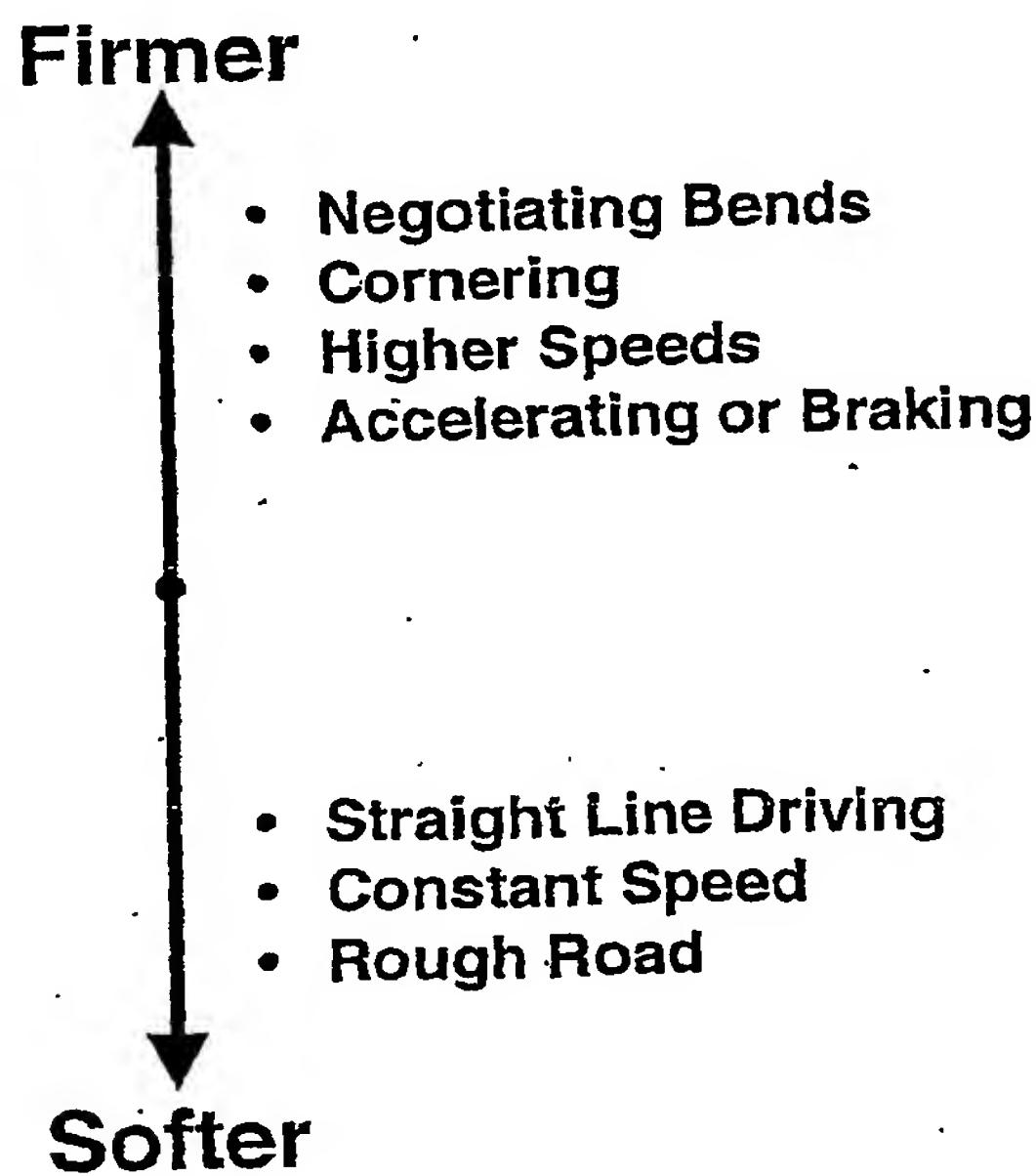


Figure 4

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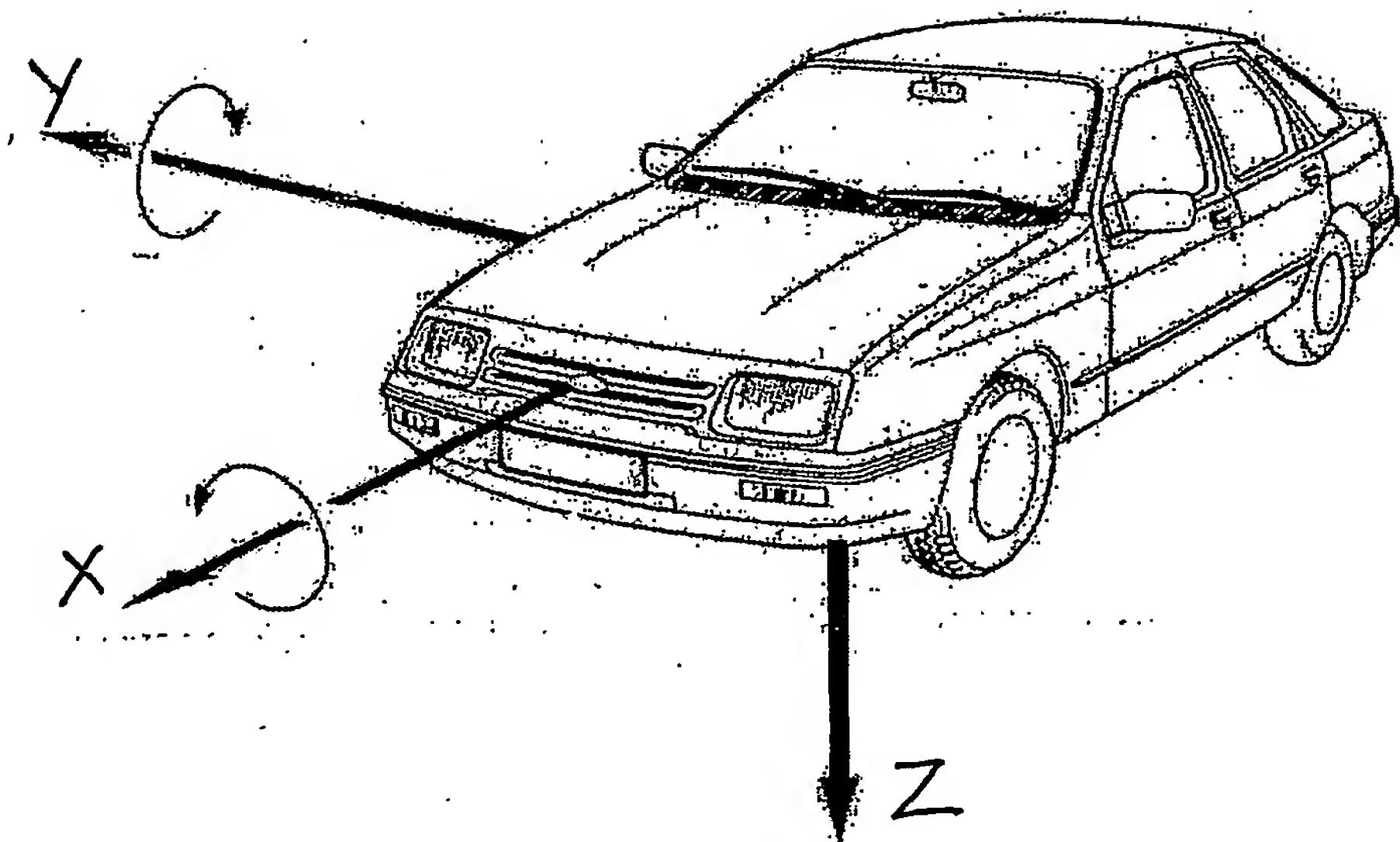


Figure 5

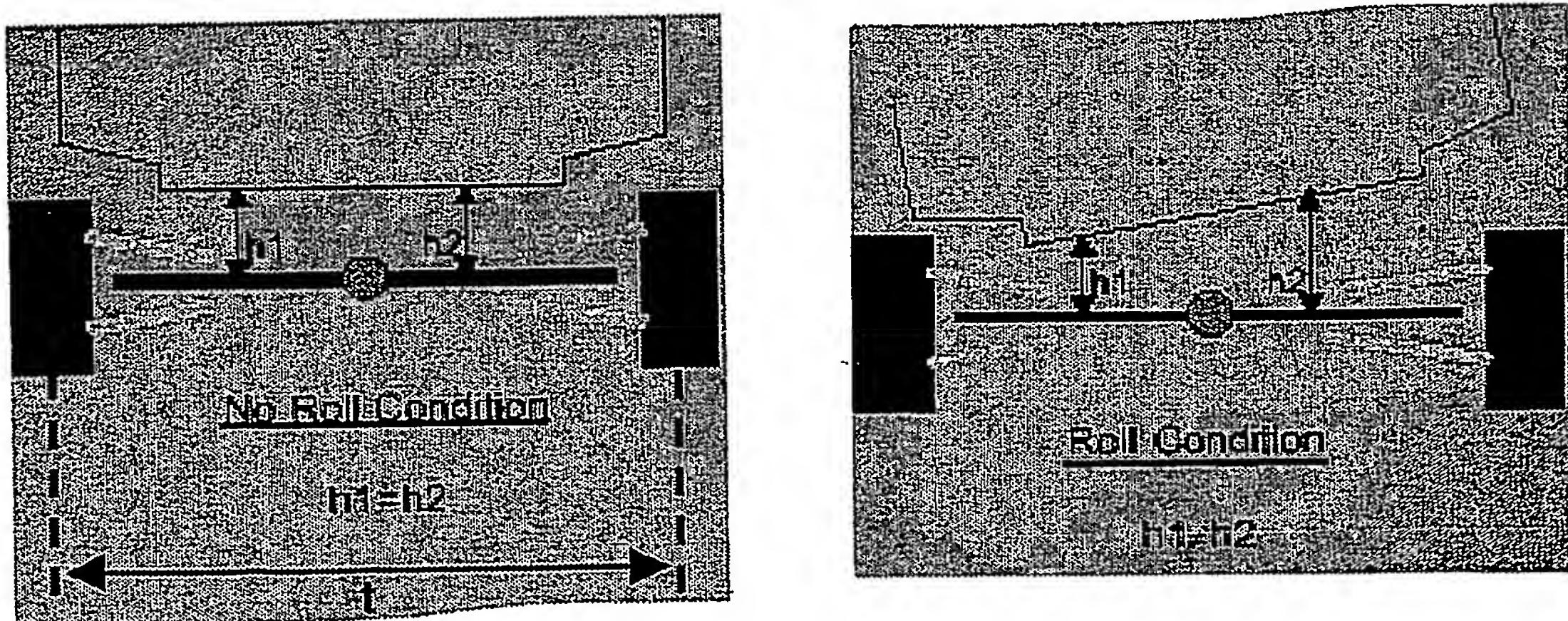


Figure 6

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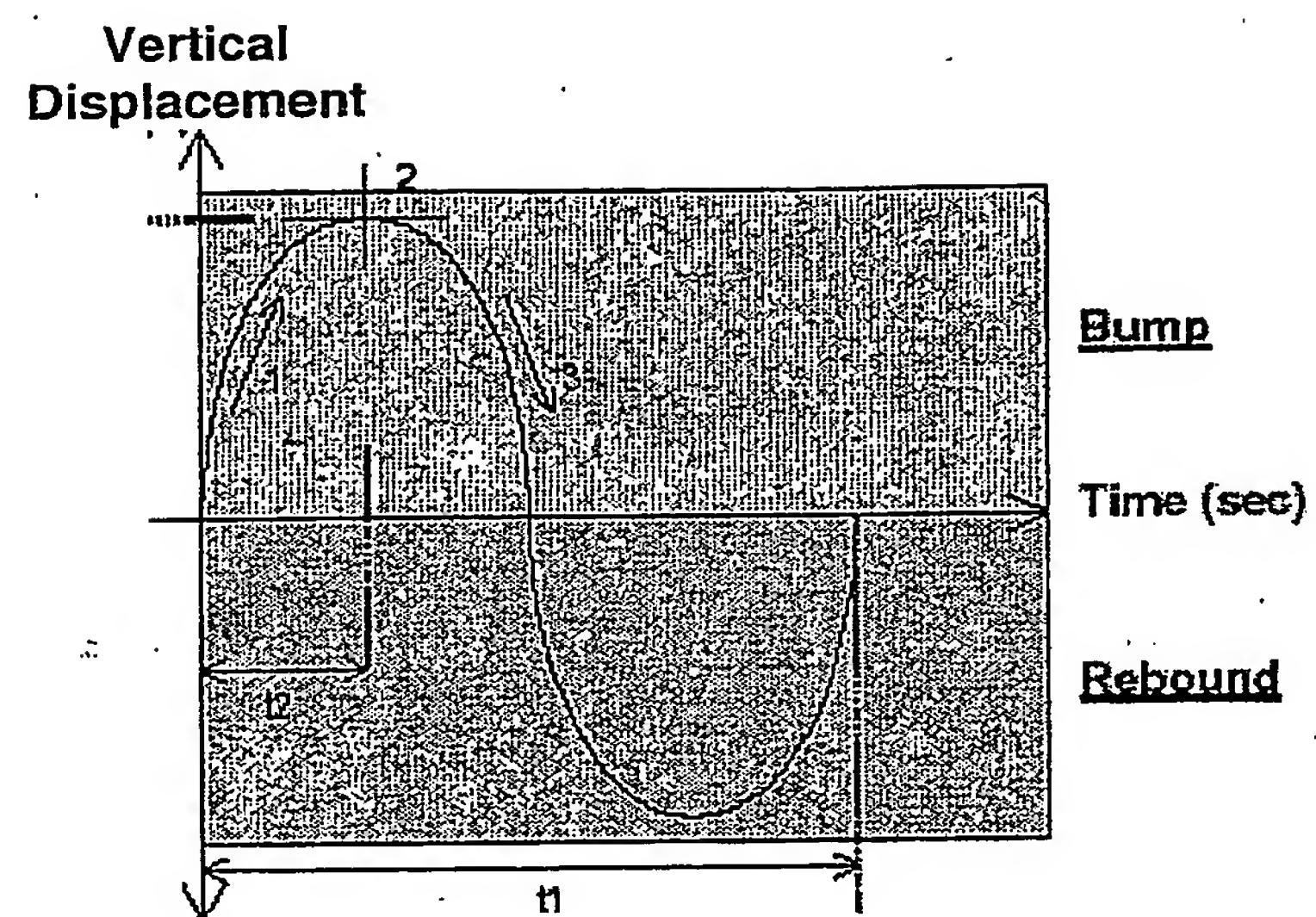


Figure 7

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